

**Program: RFEM 5, RFEM 6** 

Category: Geometrically Linear Analysis, Isotropic Linear Elasticity, Plate

Verification Example: 0063 - Centrifugal Force Loading

# 0063 - Centrifugal Force Loading

#### **Description**

A compact disc (CD) rotates at a speed 10 000 rpm. Therefore, it is subjected to the centrifugal force. The problem is modeled as a quarter model, see **Figure 1**. The problem is described by the following set of parameters.

Material	Polycarbon- ate	Modulus of Elasticity	Е	850.000	MPa
		Poisson's Ratio	ν	0.300	_
		Density	ρ	1190.000	kg/m <sup>3</sup>
Geometry		Inner radius	<i>r</i> <sub>1</sub>	7.500	mm
		Outer radius	r <sub>2</sub>	60.000	mm
		Thickness	t	1.2	mm
Load		Rotary Motion	$\omega$	1047.200	rad/s

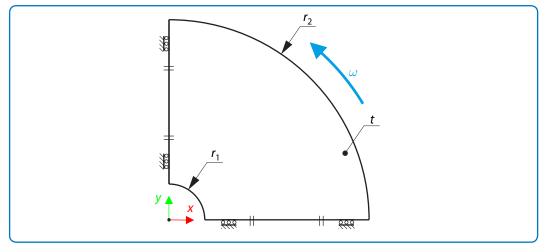


Figure 1: Problem Sketch

Determine the tangential stress  $\sigma_t$  on the inner and outer diameter at first, then determine the radial deflection of the outer radius  $u_r(r_2)$ . The self-weight is neglected.

## **Analytical Solution**

The tangential stress  $\sigma_t$  and radial stress  $\sigma_r$  on a thin rotating disc is defined as follows:

$$\sigma_{t}(r) = C_{1} + \frac{C_{2}}{r^{2}} - \frac{1 + 3\nu}{8}\rho\omega^{2}r^{2}$$
 (63 - 1)

$$\sigma_r(r) = C_1 + \frac{C_2}{r^2} - \frac{3+\nu}{8}\rho\omega^2 r^2$$
 (63 - 2)

#### Verification Example: 0063 - Centrifugal Force Loading

where  $C_1$  and  $C_2$  are real constants, which can be obtained from the boundary condition of zero radial stress  $\sigma_r$  both on the inner and outer diameter.

$$\sigma_r(r_1) = 0 \tag{63-3}$$

$$\sigma_r(r_2) = 0 \tag{63-4}$$

Therefore, constants  $C_1$  and  $C_2$  are equal to:

$$C_1 = \frac{3+\nu}{8}\rho\omega^2(r_1^2 + r_2^2) \tag{63-5}$$

$$C_2 = \frac{3+\nu}{8}\rho\omega^2 r_1^2 r_2^2 \tag{63-6}$$

The tangential stress  $\sigma_t$  on the inner and outer diameter is calculated according to formula (63 – 1).

$$\sigma_t(r_1) = 3.889 \,\text{MPa}$$
 (63 – 7)

$$\sigma_t(r_2) = 0.883 \text{ MPa}$$
 (63 – 8)

The radial deflection of the outer radius can be calculated using the Hooke's Law. Note that the radial stress on the outer radius is equal to zero.

$$u_r(r_2) = \frac{r_2}{E} \left[ \sigma_t(r_2) - \nu \sigma_r(r_2) \right] = 0.0623 \text{ mm}$$
 (63 – 9)

## **RFEM Settings**

- Modeled in RFEM 5.06 and RFEM 6.01
- The element size is  $I_{FE} = 0.001 \text{ m}$
- The number of increments is 5
- Isotropic linear elastic material model is used
- Kirchhoff plate bending theory is used

#### **Results**

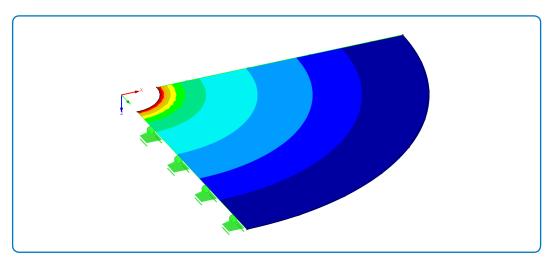


Figure 2: Results in RFEM - von Mises stress

# **Verification Example:** 0063 – Centrifugal Force Loading

Structure Files	Program
0063.01	RFEM 5, RFEM 6

Quantity	Analytical Solution	RFEM 5	Ratio	RFEM 6	Ratio
$\sigma_t(r_1)$ [MPa]	3.889	3.891	1.001	3.891	1.001
$\sigma_t(r_2)$ [MPa]	0.883	0.882	0.999	0.882	0.999
$u_r(r_2)$ [mm]	0.0623	0.0623	1.000	0.0623	1.000