

Program: RFEM 5, RF-DYNAM Pro

Category: Isotropic Linear Elasticity, Dynamics, Plate

Verification Example: 0107 – Natural Vibrations of Rectangular Membrane

0107 – Natural Vibrations of Rectangular Membrane

Description

A rectangular membrane is tensioned by a line force N according to **Figure 1**. Determine the natural frequencies of the given membrane. The problem is described by the following parameters.

Material	Steel	Modulus of Elasticity	E	210000.0	MPa
		Poisson's Ratio	ν	0.300	—
		Density	ρ	7850.000	kgm^{-3}
Geometry	Width	a	1.000	m	
	Length	b	1.500	m	
	Thickness	h	0.001	m	
Load	Tension Force	N	10000.000	Nm^{-1}	

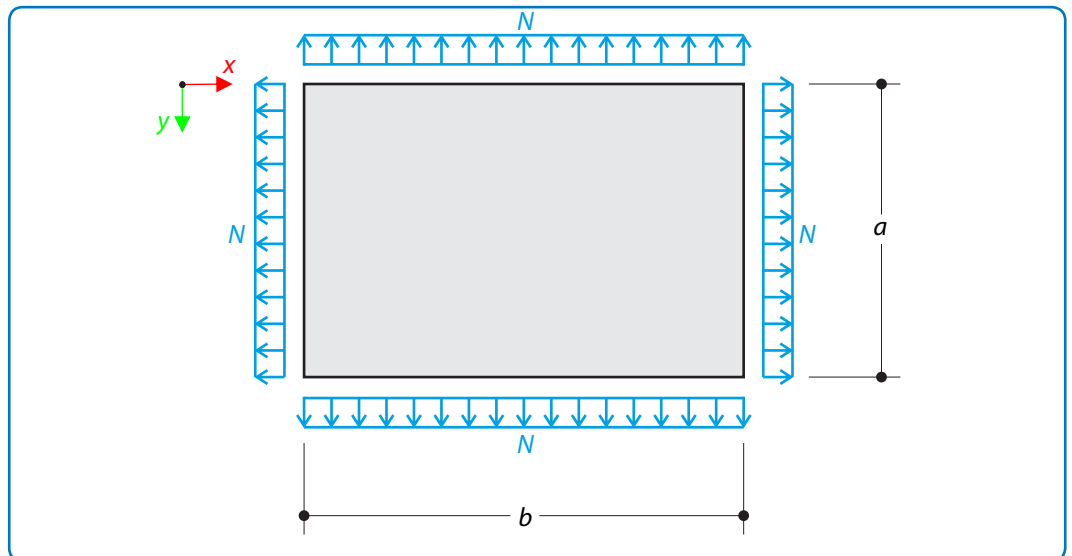


Figure 1: Problem Sketch

Analytical Solution

Free vibrations of a rectangular membrane are described by the wave equation in the following form

$$\frac{\partial^2 u_z}{\partial x^2}(x, y, t) + \frac{\partial^2 u_z}{\partial y^2}(x, y, t) - \frac{1}{c^2} \frac{\partial^2 u_z}{\partial t^2}(x, y, t) = 0 \quad (107 - 1)$$

The speed of the wave propagation c is given by the density of the membrane ρ , the membrane thickness h and the tension force N

Verification Example: 0107 – Natural Vibrations of Rectangular Membrane

$$c = \sqrt{\frac{N}{\rho h}} \quad (107 - 2)$$

The solution is sought for through separation of variables

$$u_z(x, y, t) = X(x)Y(y)T(t) \quad (107 - 3)$$

Hence, substituting into (107 - 1) yields¹

$$\frac{\ddot{T}}{T} = -\Omega^2 = c^2 \left(\frac{X''}{X} + \frac{Y''}{Y} \right) \quad (107 - 4)$$

The left-hand side depends on time t , while the right-hand side only on the spatial coordinates x and y . Thus, both sides have to be equal to a constant, which can be shown to be negative, $-\Omega^2$, for some $\Omega > 0$.

The first part of (107 - 4)

$$\ddot{T} + \Omega^2 T = 0 \quad (107 - 5)$$

yields a solution in the following form

$$T(t) = A \sin(\Omega t) + B \cos(\Omega t) \quad (107 - 6)$$

where A, B depend on the initial conditions. It follows from the second part of (107 - 4)

$$\frac{X''}{X} + \frac{Y''}{Y} = -\frac{\Omega^2}{c^2} \quad (107 - 7)$$

that the left-hand side has to be the sum of two negative constants $-\alpha^2$ and $-\beta^2$, as the two terms contain variables that are independent from each other, more precisely,

$$-\alpha^2 - \beta^2 = -\frac{\Omega^2}{c^2} \quad (107 - 8)$$

Thus, the following two equations have to be solved

$$X'' + \alpha^2 X = 0 \quad (107 - 9)$$

$$Y'' + \beta^2 Y = 0 \quad (107 - 10)$$

¹ The dashed notation indicates the derivative with respect to the space coordinate $X'' = \frac{d^2 X(x)}{dx^2}$, while the dotted notation with respect to time t .

Verification Example: 0107 – Natural Vibrations of Rectangular Membrane

The solutions are analogous to the time-variable case

$$X(x) = C \sin(\alpha x) + D \cos(\alpha x) \quad (107 - 11)$$

$$Y(y) = E \sin(\beta y) + F \cos(\beta y) \quad (107 - 12)$$

and the constants C, D, E, F depend on the boundary conditions, which, in this case, state that the deflection on the whole boundary is equal to zero

$$X(0) = 0 \quad (107 - 13)$$

$$X(a) = 0 \quad (107 - 14)$$

$$Y(0) = 0 \quad (107 - 15)$$

$$Y(b) = 0 \quad (107 - 16)$$

Using these boundary conditions, the constants D and F have to be zero. The other two boundary conditions yield

$$C \sin(\alpha a) = 0 \quad (107 - 17)$$

$$E \sin(\beta b) = 0 \quad (107 - 18)$$

which means that α, β are the roots of the sine function, more precisely

$$\alpha_m = \frac{m\pi}{a}, \quad m = 1, 2, 3, \dots \quad (107 - 19)$$

$$\beta_n = \frac{n\pi}{b}, \quad n = 1, 2, 3, \dots \quad (107 - 20)$$

Considering that $\Omega_{mn} = 2\pi f_{mn}$, when these solutions are substituted into **(107 – 8)** the natural frequencies of the rectangular membrane can be calculated according to the following equation

$$f_{mn} = \frac{c}{2} \sqrt{\frac{m^2}{a^2} + \frac{n^2}{b^2}} \quad (107 - 21)$$

For the first six natural frequencies, for $mn = 11, 12, 21, 13, 22, 23$, of the given membrane see the result table below.

RFEM 5 Settings

- Modeled in RFEM 5.16.01
- The element size is $l_{FE} = 0.030$ m
- Isotropic linear elastic material model is used

Verification Example: 0107 – Natural Vibrations of Rectangular Membrane

Results

Structure Files	Program		
0107.01	RFEM 5 – RF-DYNAM Pro		
Quantity	Analytical Solution	RFEM 5	Ratio
f_1 [Hz]	21.448	21.441	1.000
f_2 [Hz]	29.743	29.726	0.999
f_3 [Hz]	37.622	37.573	0.999
f_4 [Hz]	39.904	39.854	0.999
f_5 [Hz]	42.896	42.844	0.999
f_6 [Hz]	50.475	50.401	0.999

Figure 2 shows the first six natural shapes of the investigated membrane.

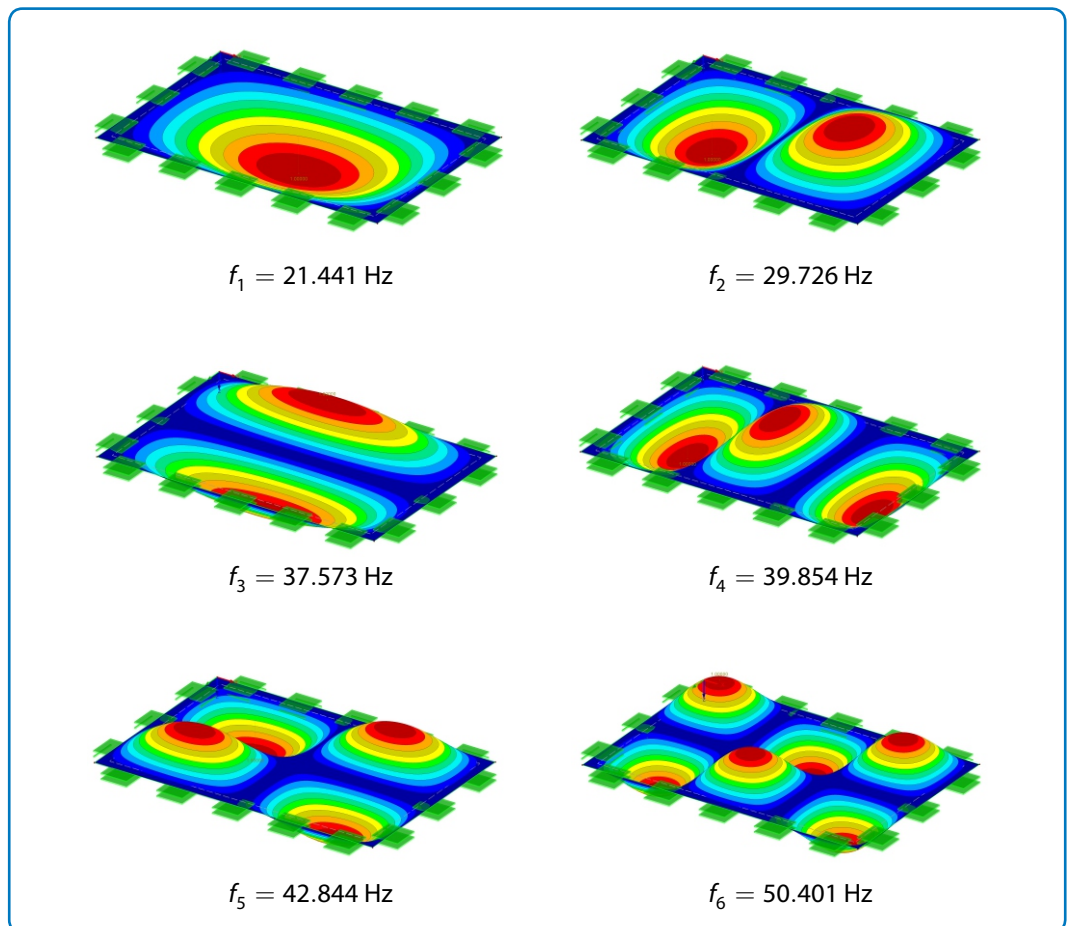


Figure 2: First six natural shapes of the membrane in RFEM 5