

**Program:** RFEM 5, RSTAB 8, RF-STEEL, STEEL

**Category:** Geometrically Linear Analysis, Isotropic Linear Elasticity, Member

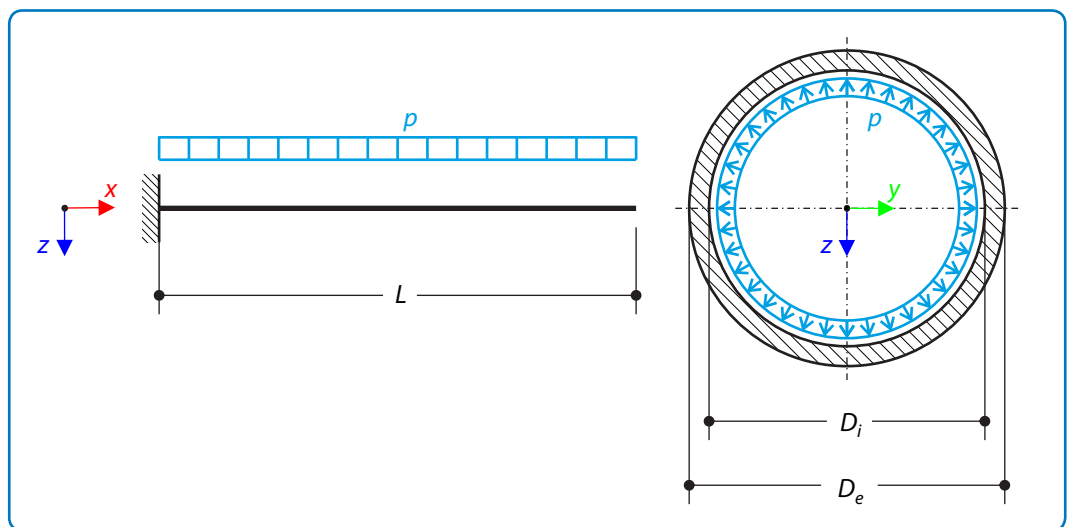
**Verification Example:** 0089 – Bourdon Effect

## 0089 – Bourdon Effect

### Description

A pipe with the tubular cross-section is loaded by means of internal pressure, see **Figure 1**. The internal pressure causes axial deformation of the pipe, what is called Bourdon effect. Determine the axial deformation  $u_x$  of the pipe endpoint. The problem is described by the following parameters.

Material	Steel	Modulus of Elasticity	$E$	210000.000	MPa
		Poisson's Ratio	$\nu$	0.300	—
Geometry	Pipe	Length	$L$	10.000	m
		Outer Diameter	$D_e$	200.000	mm
		Inner Diameter	$D_i$	196.000	mm
Load		Pipe Internal Pressure	$p$	1.000	MPa



**Figure 1:** Problem Sketch

### Analytical Solution

The pipe is considered to be a thick-walled close-ended vessel. Detailed description of the thick-walled vessel calculation can be found in Verification example 0064, [1]. The stress state of the pipe is generally spatial due to the radial stress  $\sigma_r$ , tangential stress  $\sigma_t$  and axial stress  $\sigma_x$ . The axial deformation  $u_x$  of the pipe endpoint is defined by means of Hooke's law

$$u_x = \frac{L}{E} (\sigma_x - \nu(\sigma_r + \sigma_t)). \quad (89 - 1)$$

### Verification Example: 0089 – Bourdon Effect

The axial stress  $\sigma_x$  when considering zero outer pressure is

$$\sigma_x = \frac{pr_i^2}{r_e^2 - r_i^2}, \quad (89 - 2)$$

where  $r_e$  and  $r_i$  is the outer and inner radius respectively. The radial and tangential stresses are defined as follows

$$\sigma_r = \sigma_x - \frac{C}{r^2}, \quad (89 - 3)$$

$$\sigma_t = \sigma_x + \frac{C}{r^2}, \quad (89 - 4)$$

where  $r$  is the radial coordinate and  $C$  is the real constant, in this case it is equal to

$$C = p \frac{r_i^2 r_e^2}{r_e^2 - r_i^2}. \quad (89 - 5)$$

Due to the agreement with RFEM 5 / RSTAB 8 analysis, further calculations are carried out for the middle radius  $r_m = \frac{r_e + r_i}{2}$ . Using (89 - 1), the axial deformation  $u_x$  of the pipe endpoint results

$$u_x \approx 0.462 \text{ mm}. \quad (89 - 6)$$

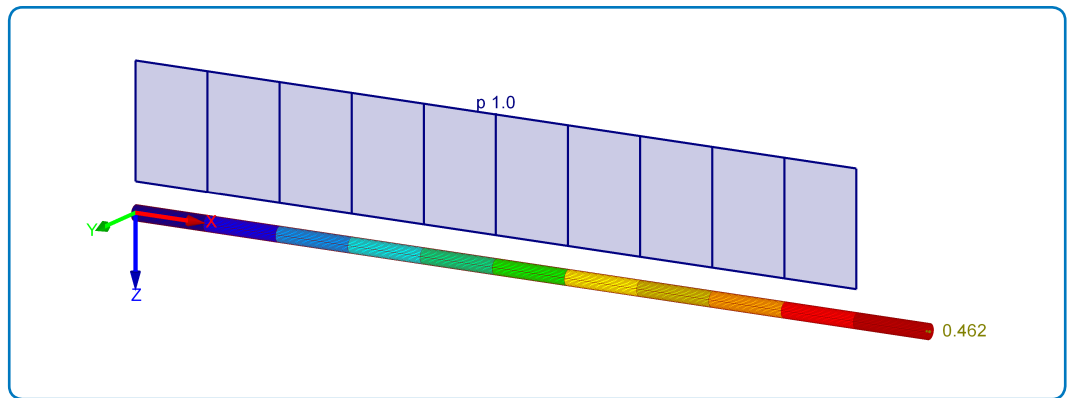
### RFEM 5 and RSTAB 8 Settings

- Modeled in RFEM 5.16.01 and RSTAB 8.16.01
- Element size is  $l_{FE} = 0.500 \text{ m}$
- Isotropic linear elastic material is used
- Member load 'Pipe internal pressure' is used
- Displacements due to the member loads of type 'Pipe internal pressure' (Bourdon effect) are enabled
- To obtain the values of the stresses including reduced Von Mises stress it is necessary to use add-on module RF-STEEL or STEEL with appropriate standard

### Results

Structure File	Program
0089.01	RFEM 5
0089.02	RSTAB 8

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**Figure 2:** Results

Quantity	Analytical Solution	RFEM 5	Ratio	RSTAB 8	Ratio
$u_x$ [mm]	0.462	0.462	1.000	0.462	1.000

## References

[1] DLUBAL SOFTWARE GMBH, *Verification Example 0064 – Thick-Walled Vessel*. 2016.