#### Program: RFEM 5, RSTAB 8, RF-DYNAM Pro

**Category:** Large Deformation Analysis, Post-Critical Analysis, Isotropic Linear Elasticity, Dynamics, Member

Verification Example: 0045 – Snap-Through

## 0045 – Snap-Through

## Description

A structure is made of two trusses, which are embedded into the hinge supports according to the **Figure 1**. The structure is loaded by the concentrated force  $F_z$ . The self-weight is neglected in this example. Determine the relationship between the loading force  $F_z$  and the deflection  $u_z$  considering large deformations generally. Determine the deflection under the loading force  $F_z = 122000$  kN of the connection point of the trusses. The problem is described by the following set of parameters.

Material	Steel	Modulus of Elasticity	E	210000.000	MPa
		Poisson's Ratio	ν	0.300	-
Geometry	Structure	Truss Length	L <sub>0</sub>	3.000	m
		Height	h	1.500	m
	Cross-Section	Width	а	100.000	mm
Load		Force	Fz	122000.000	kN

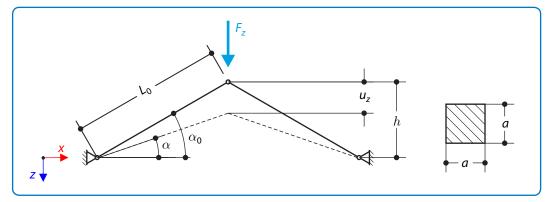


Figure 1: Problem sketch

## **Analytical Solution**

The axial force *N* in the truss can be determined from the force equilibrium according to the **Figure 2**.

$$N = \frac{F_z}{2\sin\alpha} \tag{45-1}$$

Considering Geometrically linear analysis the angle  $\alpha$  is remaining constant  $\alpha = \alpha_0$  and the formula (45 – 1) can be rewritten into the following form.



#### Verification Example: 0045 – Snap-Through

$$N = F_z \frac{L_0}{2h} \tag{45-2}$$

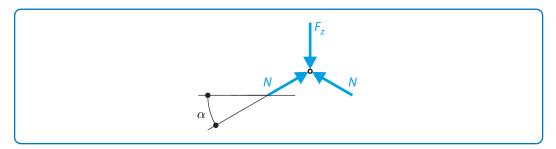


Figure 2: Force equilibrium

Using the Geometrically linear analysis the axial force *N* is not changing during the deformation of the structure. This is not corresponding with the expectations. The large deformation analysis should be used instead. The axial deformation of the truss can be then determined as follows.

$$\Delta L = L - L_0 = \sqrt{(h - u_z)^2 + b^2} - L_0$$
(45 - 3)

Where *b* is the half width of the structure and *L* is the length of the truss after the deformation. It can be calculated as follows.

$$b = \sqrt{L_0^2 - h^2} \tag{45-4}$$

The formula (45 – 1) can be improved into the following form.

$$N = F_z \frac{L}{2(h - u_z)} = F_z \frac{\sqrt{(h - u_z)^2 + b^2}}{2(h - u_z)}$$
(45 - 5)

The axial force N can be also determined from the Hooke's law<sup>1</sup> as

$$N = \varepsilon EA \tag{45-6}$$

Considering the large deformation analysis the logarithmic form of the axial strain  $\varepsilon$  should be used.

$$\varepsilon = \ln\left(1 - \frac{\Delta L}{L_0}\right) \tag{45-7}$$

Using above mentioned formulae the general relationship between loading force  $F_z$  and the deflection  $u_z$  can be determined.



<sup>&</sup>lt;sup>1</sup> Hooke's law  $\sigma = E\varepsilon$ . The axial stress is defined as  $\sigma = \frac{N}{A}$ , where A is the cross-section area.

#### Verification Example: 0045 – Snap-Through

$$F_{z} = \frac{2EA(h - u_{z})\ln\left(1 - \frac{\sqrt{(h - u_{z})^{2} + b^{2}} - L_{0}}{L_{0}}\right)}{\sqrt{(h - u_{z})^{2} + b^{2}}}$$
(45 - 8)

The formula (45 – 8) is nonlinear. The relationship between loading force  $F_z$  and the deflection  $u_z$  is shown in **Figure 3**. There is also shown the comparison with numerical solutions.

# **RSTAB 8 and RFEM 5 Settings**

- Modeled in RSTAB 8.26 and RFEM 5.26
- The element size is  $I_{\rm FE}=0.025~{
  m m}$
- The number of increments is 10
- The structure is modeled using members (Truss only N)
- Shear stiffness of the members is neglected
- Isotropic linear elastic material model is used
- In global calculation parameters there is disabled: Activate member divisions for large deformation or post-critical analysis

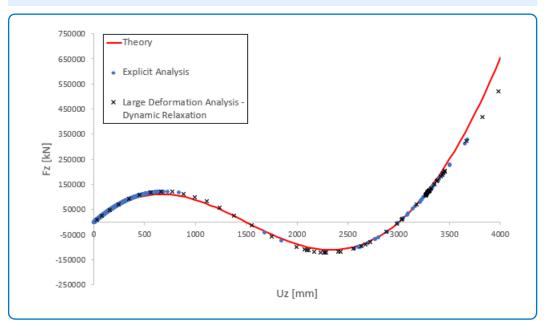
### **Results**

Structure Files	Program	Solving Method	
0045.01	RFEM 5	Post-Critical Analysis – Modified Newton-Raphson	
0045.02	RFEM 5	Large Deformation Analysis – Dynamic Relaxation	
0045.03	RSTAB 8	Post-Critical Analysis – Modified Newton-Raphson	
0045.04	RFEM 5 – RF-DYNAM Pro	Explicit Analysis	

Model	Analytical Solution	RSTAB 8 and RFEM 5 Solution	
	<i>u<sub>z</sub></i> [m]	<i>u<sub>z</sub></i> [m]	Ratio [-]
RFEM 5 (Modified Newton-Raphson)		3.294	1.004
RFEM 5 (Dynamic Relaxation)	3.282	3.294	1.004
RSTAB 8 (Modified Newton-Raphson)	3.262	3.288	1.002
RFEM 5 – RF-DYNAM Pro (Explicit Analysis)		3.294	1.004



# Verification Example: 0045 – Snap-Through



**Figure 3:** Comparison of the theoretical solution, explicit analysis and large deformation analysis - dynamic relaxation

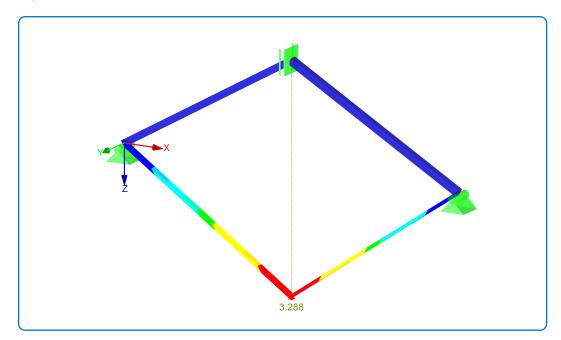


Figure 4: RFEM 5 / RSTAB 8 Results

