

**Program:** RFEM 5, RSTAB 8, RF-DYNAM Pro, DYNAM Pro

**Category:** Geometrically Linear Analysis, Dynamics, Member

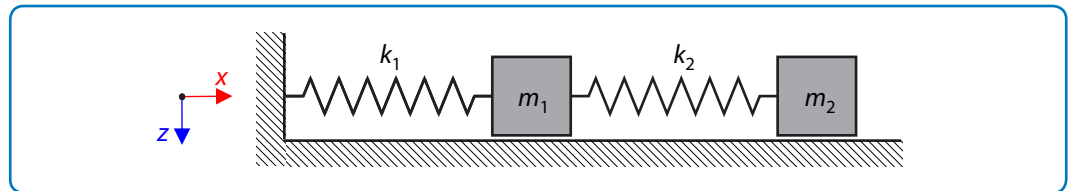
**Verification Example:** 0117 – Double Mass Oscillator

## 0117 – Double Mass Oscillator

### Description

A double-mass oscillator consists of two linear springs with stiffness  $k_1, k_2$  and masses  $m_1, m_2$ , which are concentrated at the nodes. The self-weight of the springs is neglected. Determine the natural frequencies of the system. The problem is shown in **Figure 1** and is described by the following set of parameters.

System Properties	Mass	$m_1$	50.000	kg
		$m_2$	20.000	kg
	Spring Stiffness	$k_1$	10.000	kN/m
		$k_2$	20.000	kN/m



**Figure 1:** Problem Sketch

### Analytical Solution

Natural oscillations of the double-mass system are described by the following system of second-order differential equations

$$m_1 \ddot{u}_{x1} + k_1 u_{x1} - k_2 (u_{x2} - u_{x1}) = 0, \quad (117 - 1)$$

$$m_2 \ddot{u}_{x2} + k_2 (u_{x2} - u_{x1}) = 0, \quad (117 - 2)$$

or in matrix form

$$\mathbf{M} \ddot{\mathbf{u}}_x + \mathbf{K} \mathbf{u}_x = 0, \quad (117 - 3)$$

$$\begin{bmatrix} m_1 & 0 \\ 0 & m_2 \end{bmatrix} \begin{bmatrix} \ddot{u}_{x1} \\ \ddot{u}_{x2} \end{bmatrix} + \begin{bmatrix} k_1 + k_2 & -k_2 \\ -k_2 & k_2 \end{bmatrix} \begin{bmatrix} u_{x1} \\ u_{x2} \end{bmatrix} = 0. \quad (117 - 4)$$

The eigenvalue problem

$$\det(\mathbf{K} - \Omega^2 \mathbf{M}) = 0 \quad (117 - 5)$$

then yields the angular frequencies  $\Omega_1$  and  $\Omega_2$ , whence the natural frequencies  $f_1, f_2$  are, in turn, calculated.

## Verification Example: 0117 – Double Mass Oscillator

$$f_1 = \frac{\Omega_1}{2\pi} \approx 1.861 \text{ Hz} \quad (117 - 6)$$

$$f_2 = \frac{\Omega_2}{2\pi} \approx 6.088 \text{ Hz} \quad (117 - 7)$$

### RFEM 5 and RSTAB 8 Settings

- Modeled in RFEM 5.10.00 and RSTAB 8.10.00
- Subspace iteration method is used

### Results

Structure Files	Program
0117.01	RFEM 5 – RF-DYNAM Pro
0117.02	RSTAB 8 – DYNAM Pro

Model	Analytical Solution $f_1$ [Hz]	RFEM 5 / RSTAB 8	
		$f_1$ [Hz]	Ratio [-]
RFEM 5	1.861	1.861	1.000
RFEM 8		1.862	1.000

Model	Analytical Solution $f_2$ [Hz]	RFEM 5 / RSTAB 8	
		$f_2$ [Hz]	Ratio [-]
RFEM 5	6.088	6.088	1.000
RSTAB 8		6.088	1.000