

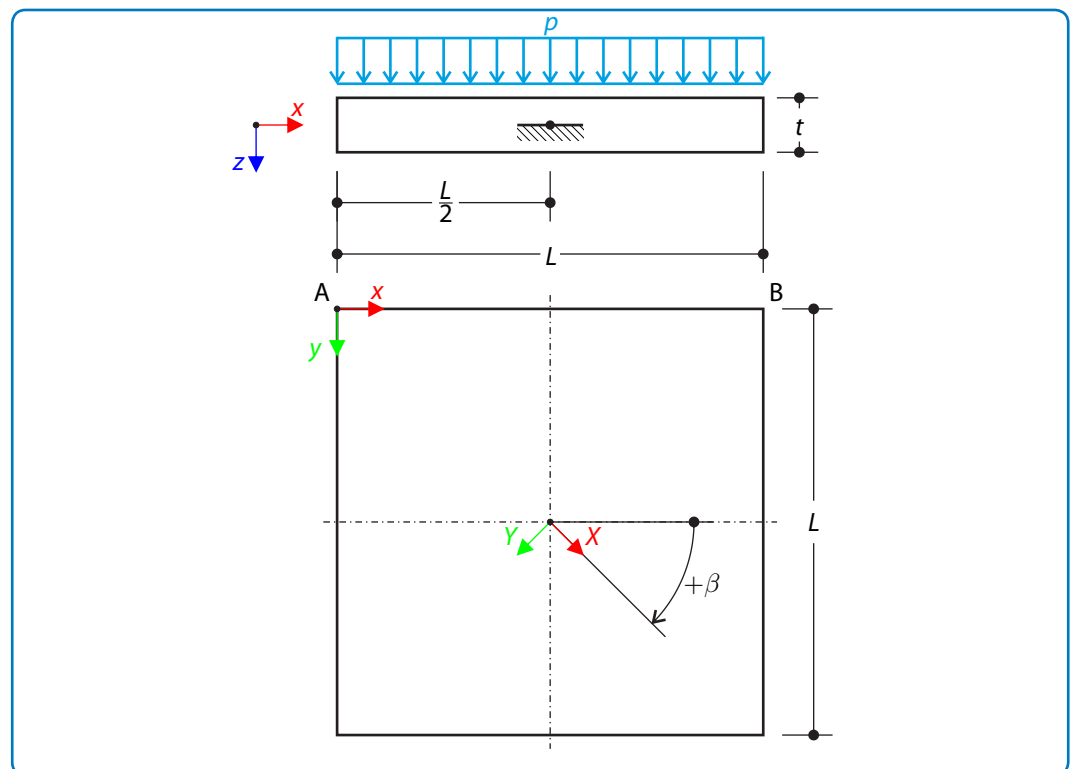
## 0029 – Fiber Rotation Test in Laminated Plates

### Description

One layered square orthotropic plate of side length  $L$  and thickness  $t$  is fully fixed at its middle point and subjected to the pressure  $p$  according to the **Figure 1**. Problem is described by the following set of parameters.

Material	Laminate	Modulus of Elasticity	$E_X$	8000.000	MPa
			$E_Y = E_Z$	270.000	MPa
		Poisson's Ratio	$\nu_{YZ}$	0.350	–
			$\nu_{XY} = \nu_{XZ}$	0.470	–
		Shear Modulus	$G_{XY} = G_{XZ}$	500.000	MPa
			$G_{YZ}$	100.000	MPa
Geometry		Side Length	$L$	10.000	m
		Thickness	$t$	0.100	m
		Fibers Angle	$\beta$	$\pm 45$	°
Load		Pressure	$p$	1.000	Pa

Compare the deflection  $u_z$  of the plate corners A (0, 0, 0) and B ( $L$ , 0, 0) for different fiber angles  $\beta$  to check the correctness of the transformation.



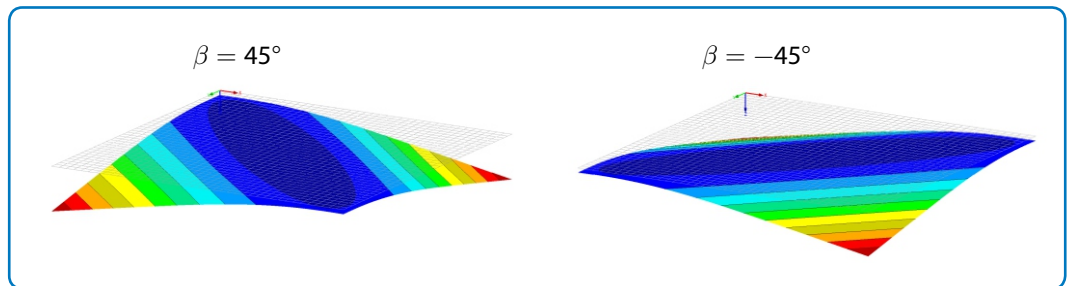
**Figure 1:** Problem sketch

### Analytical Solution

The aim of this test is to prove the equality of the deflections in opposite corners for fibre orientation  $\beta = \pm 45^\circ$ . Complete analytical solution is not available, because of the given boundary conditions. The reason of those boundary conditions is the suitable comparability of the results for fiber orientation angle  $\beta = \pm 45^\circ$ . The stiffness of the plate is much greater in fibres direction (direction of X-axis). The different fiber orientation causes the different stiffness of the plate in directions of diagonals, see **Figure 2**. The plate is more stiffer in the direction of the diagonal which is parallel to the fibre direction. Thus it is possible to write

$$u_{z,A} > u_{z,B}, \quad \beta = 45^\circ \quad (29 - 1)$$

$$u_{z,A} < u_{z,B}, \quad \beta = -45^\circ \quad (29 - 2)$$



**Figure 2:** Different stiffness of the diagonals for fibre orientation  $\beta = \pm 45^\circ$

### Stiffness Matrix

The stiffness matrix elements calculation follows. The global stiffness matrix can be calculated analytically. The stiffness matrix for one layer in local coordinates is defined as follows

$$\mathbf{d}' = \begin{bmatrix} \frac{E_X}{1 - \nu_{XY}\nu_{YX}} & \frac{\nu_{XY}E_Y}{1 - \nu_{XY}\nu_{YX}} & 0 \\ \frac{\nu_{YX}E_X}{1 - \nu_{XY}\nu_{YX}} & \frac{E_Y}{1 - \nu_{XY}\nu_{YX}} & 0 \\ \text{sym.} & & G_{XY} \end{bmatrix} \quad (29 - 3)$$

The relationship between moduli of elasticity and Poisson's ratios for the orthotropic materials is defined as follows:

$$\frac{\nu_{YX}}{E_Y} = \frac{\nu_{XY}}{E_X} \quad (29 - 4)$$

For the transformation of the local stiffness matrix  $\mathbf{d}'$  to the global coordinate system by rotation by an angle  $\beta$  the transformation matrix  $\mathbf{T}$  is used

$$\mathbf{d} = \mathbf{T}^T \mathbf{d}' \mathbf{T} \quad (29 - 5)$$

The transformation matrix is defined as

$$\mathbf{T} = \begin{bmatrix} \cos^2 \beta & \sin^2 \beta & \cos \beta \sin \beta \\ \sin^2 \beta & \cos^2 \beta & -\cos \beta \sin \beta \\ -2 \cos \beta \sin \beta & 2 \cos \beta \sin \beta & \cos^2 \beta - \sin^2 \beta \end{bmatrix} \quad (29 - 6)$$

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The stiffness matrix for one layer then results for  $\beta = \pm 45^\circ$ :

$$\mathbf{d} = \begin{bmatrix} 2.647 & 1.647 & \pm 1.947 \\ 1.647 & 2.647 & \pm 1.947 \\ \pm 1.947 & \pm 1.947 & 2.019 \end{bmatrix} \text{ GPa} \quad (29 - 7)$$

Global stiffness matrix has the following form:

$$\mathbf{D} = \begin{bmatrix} D_{11} & D_{12} & D_{13} & 0 & 0 & D_{16} & D_{17} & D_{18} \\ & D_{22} & D_{23} & 0 & 0 & & D_{27} & D_{28} \\ & & D_{33} & 0 & 0 & \text{sym.} & & D_{38} \\ & & & D_{44} & D_{45} & 0 & 0 & 0 \\ & & & & D_{55} & 0 & 0 & 0 \\ \text{sym.} & & & & & D_{66} & D_{67} & D_{68} \\ & & & & & & D_{77} & D_{78} \\ & & & & & & & D_{88} \end{bmatrix} \quad (29 - 8)$$

Elements of the stiffness matrix  $D_{11} - D_{33}$  define bending and torsion. General formula for these elements can be written as follows

$$D_{ij} = \sum_{k=1}^n \frac{z_{k,\max}^3 - z_{k,\min}^3}{3} d_{k,ij} \quad (29 - 9)$$

where  $i = 1, 2, 3; j = 1, 2, 3$  and  $n$  defines the number of the layers,  $z_{k,\max}$  and  $z_{k,\min}$  corresponds to the maximum and minimum distance of the appropriate layer surfaces from the zero layer ( $z = 0$ ). In this special case with only one layer of the thickness  $t$  the elements can be calculated as follows.

$$D_{11} = \frac{t^3}{12} d_{11} = 220.580 \text{ kNm} \quad (29 - 10)$$

$$D_{12} = \frac{t^3}{12} d_{12} = 137.246 \text{ kNm} \quad (29 - 11)$$

$$D_{13} = \frac{t^3}{12} d_{13} = \pm 162.251 \text{ kNm} \quad (29 - 12)$$

$$D_{22} = \frac{t^3}{12} d_{22} = 220.580 \text{ kNm} \quad (29 - 13)$$

$$D_{23} = \frac{t^3}{12} d_{23} = \pm 162.251 \text{ kNm} \quad (29 - 14)$$

$$D_{33} = \frac{t^3}{12} d_{33} = 168.259 \text{ kNm} \quad (29 - 15)$$

Thanks to the symmetry (reference plane is in the middle of the layer) elements of the stiffness matrix  $D_{16} - D_{38}$  are equal to zero. The elements  $D_{44} - D_{55}$  are not taken into account due to the assumption of Kirchhoff plate bending theory (no shear effects). Elements of the stiffness matrix  $D_{66} - D_{88}$ , which define membrane loading can be calculated for angles  $\beta = \pm 45^\circ$  as follows

$$D_{i+5,j+5} = \sum_{k=1}^n (z_{k,\max} - z_{k,\min}) d_{k,ij} \quad (29 - 16)$$

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where  $i = 1, 2, 3; j = 1, 2, 3$  and  $n$  defines the number of the layers,  $z_{k,\max}$  and  $z_{k,\min}$  corresponds to the maximum and minimum distance of the appropriate layer surfaces from the zero layer ( $z = 0$ ). In this special case with only one layer of the thickness  $t$  the elements can be calculated as follows.

$$D_{66} = td_{11} = 264696 \text{ kNm} \quad (29 - 17)$$

$$D_{67} = td_{12} = 164696 \text{ kNm} \quad (29 - 18)$$

$$D_{68} = td_{13} = \pm 194702 \text{ kNm} \quad (29 - 19)$$

$$D_{77} = td_{22} = 264696 \text{ kNm} \quad (29 - 20)$$

$$D_{78} = td_{23} = \pm 194702 \text{ kNm} \quad (29 - 21)$$

$$D_{88} = td_{33} = 201910 \text{ kNm} \quad (29 - 22)$$

### RFEM Settings

- Modeled in RFEM 5.26 and RFEM 6.01
- The element size is  $l_{FE} = 0.250 \text{ m}$
- Geometrically linear analysis is considered
- The number of increments is 5
- Kirchhoff plate bending theory is used
- Orthotropic Elastic 2D material model is used

### Results

Structure File	Program	Fiber Orientation
0029.01	RFEM 5, RFEM 6	45°
0029.02	RFEM 5, RFEM 6	-45°
0029.03	RF-LAMINATE	45°
0029.04	RF-LAMINATE	-45°

Fiber orientation $\beta = +45^\circ$	RFEM 5	RF-LAMINATE	RFEM 6
Test point	$u_z$ [mm]	$u_z$ [mm]	$u_z$ [mm]
A	0.502	0.502	0.502
B	4.976	4.976	4.976

**Verification Example: 0029 – Fiber Rotation Test in Laminated Plates**

Fiber orientation $\beta = -45^\circ$	RFEM 5	RF-LAMINATE	RFEM 6
Test point	$u_z$ [mm]	$u_z$ [mm]	$u_z$ [mm]
A	4.976	4.976	4.976
B	0.502	0.502	0.502