

**Program:** RFEM 5

**Category:** Geometrically Linear Analysis, Isotropic Linear Elasticity, Plate

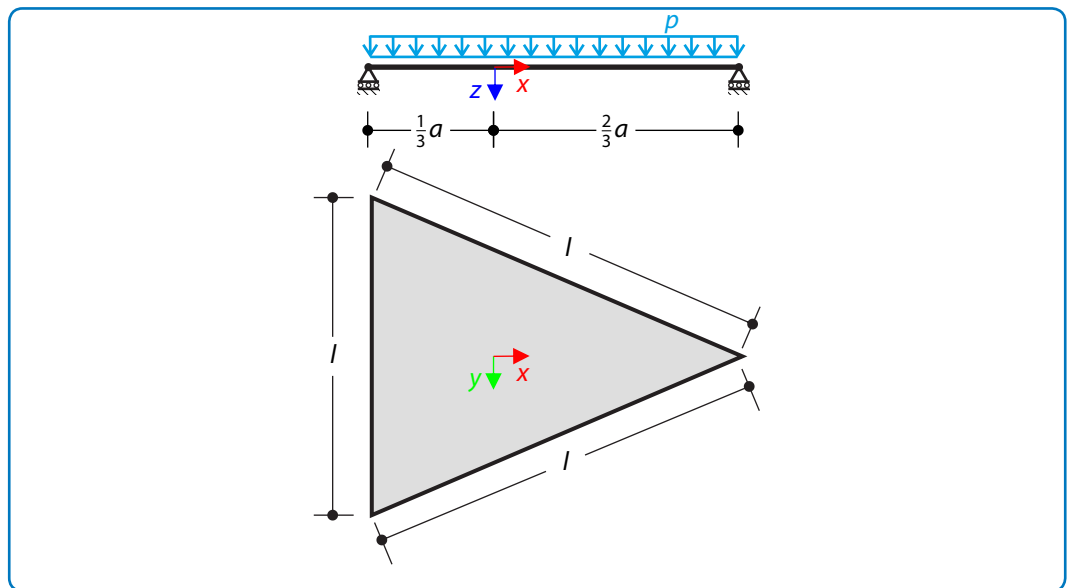
**Verification Example:** 0072 – Bending of a Simply Supported Triangular Plate

## 0072 – Bending of a Simply Supported Triangular Plate

### Description

A simply supported equilateral triangular plate is subjected to uniformly distributed transversal load  $p$ . Assuming small deformation theory and neglecting self-weight, determine the maximum out-of-plane deflection  $u_{\max}$  of the plate.

Material	Linear Elastic	Modulus of Elasticity	$E$	50.000	GPa
		Poisson's Ratio	$\nu$	0.200	—
Geometry	Triangle	Thickness	$t$	0.200	m
		Side Length	$l$	2.000	m
Load		Pressure	$p$	10.000	MPa



**Figure 1:** Problem sketch

### Analytical Solution

For a simply supported equilateral triangular plate under uniform lateral loads, a closed-form solution has been proposed by Woinowsky & Krieger and can be found, e.g., in [1]

$$u(x, y) = \frac{p}{64Da} \left( x^3 - x^2a - 3xy^2 - y^2a + \frac{4}{27}a^3 \right) \left( \frac{4}{9}a^2 - x^2 - y^2 \right) \quad (72 - 1)$$

where  $D$  is the flexural rigidity of the plate

$$D = \frac{Et^3}{12(1 - \nu^2)} \quad (72 - 2)$$

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and  $a$  the length of the triangle altitude, namely

$$a = \frac{\sqrt{3}}{2}l \quad (72 - 3)$$

The plate will be deflected the most at its orthocenter where  $x = 0$  and  $y = 0$ , as this simply supported equilateral triangle can be replaced by an "equivalent" circular plate centered at the orthocenter of the triangle with radius  $r = 0.35a$ , see [1] for the details. Hence the maximum deflection reads as

$$u_{\max} = u(0, 0) = \frac{pl^4(1 - \nu^2)}{144Et^3} \approx 2.667 \text{ mm} \quad (72 - 4)$$

### RFEM 5 Settings

- Modeled in version RFEM 5.06.3039
- The element size is  $l_{FE} = 0.01 \text{ m}$
- Geometrically linear analysis is considered
- The number of increments is 1
- Kirchhoff plate theory is used

### Results

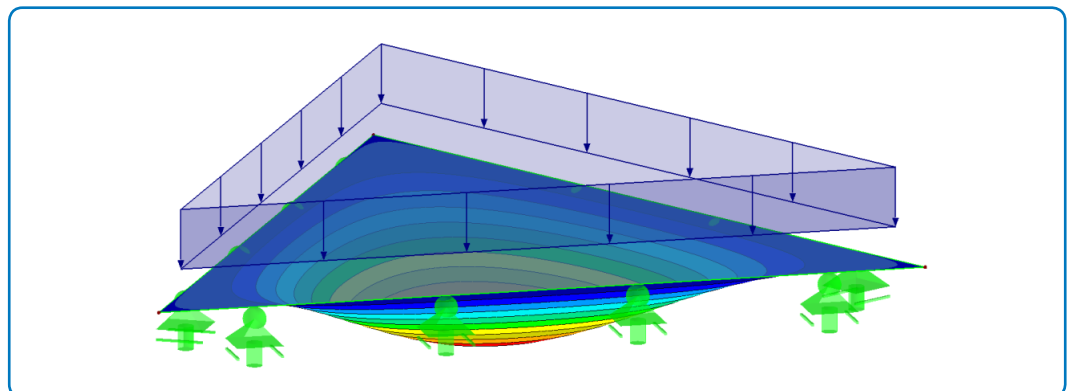


Figure 2: RFEM 5 Solution

Structure File	Program
0072.01	RFEM 5

As can be seen from the table below, excellent agreement of numerical output with the analytical result was achieved.

Analytical Solution	RFEM 5	
$u_{\max}$ [mm]	$u_{\max}$ [mm]	Ratio [-]
2.667	2.666	1.000

### References

- [1] SZILARD, R. *Theories and Application of Plate Analysis: Classical Numerical and Engineering Method*. Hoboken, New Jersey.