

**Program:** RFEM 5, RSTAB 8, RF-DYNAM Pro, DYNAM Pro

**Category:** Second-Order Analysis, Isotropic Linear Elasticity, Dynamics, Member

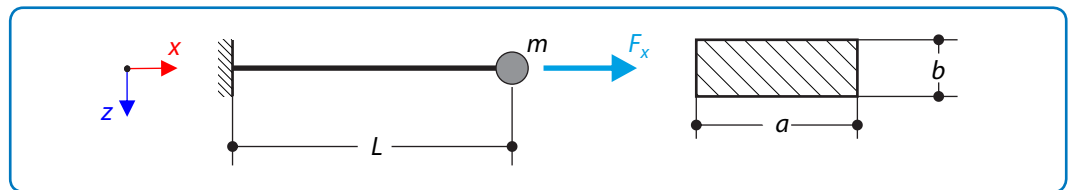
**Verification Example:** 0115 – Bending Vibrations with Axial Force

## 0115 – Bending Vibrations with Axial Force

### Description

A cantilever of rectangular cross-section  $a \times b$  has a mass  $m$  at its end. Furthermore, it is loaded by an axial force  $F_x$  according to **Figure 1**. Calculate the natural frequency of the structure. Neglect the self-weight of the cantilever and consider the influence of axial force for the stiffness modification. The problem is described by the following set of parameters.

Material	Steel	Modulus of Elasticity	$E$	210000.0	MPa
		Poisson's Ratio	$\nu$	0.300	—
Geometry	Width	$a$	0.050	m	
	Height	$b$	0.010	m	
	Length	$L$	0.500	m	
Load	Axial Force	$F_x$	1.000	kN	
	Mass	$m$	25.000	kg	



**Figure 1:** Problem Sketch

### Analytical Solution

The axial force  $F_x$  influences the bending stiffness of the cantilever. Thus the natural bending frequency is also modified. The analytical solution of the bending stiffness is based on the solution presented in Verification Examples 0042 and 0048, [1], [2]. The deflection of the cantilever considering the effect of axial force is defined as follows

$$u_z = \frac{F_z (L\alpha e^{\alpha L} + L\alpha e^{-\alpha L} - e^{\alpha L} + e^{-\alpha L})}{F_x \alpha (e^{\alpha L} + e^{-\alpha L})} \quad (115 - 1)$$

where  $F_z$  is the general transversal force, and  $\alpha$  is defined as

$$\alpha = \sqrt{\frac{F_x}{EI_y}} \quad (115 - 2)$$

The bending stiffness  $k$  evaluates then as

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$$k = \frac{F_z}{u_z} = \frac{F_x \alpha (e^{\alpha L} + e^{-\alpha L})}{L \alpha e^{\alpha L} + L \alpha e^{-\alpha L} - e^{\alpha L} + e^{-\alpha L}} \quad (115 - 3)$$

The natural frequency  $f$  of the single-mass oscillator is defined according to the well-known formula

$$f = \frac{1}{2\pi} \sqrt{\frac{k}{m}} \approx 4.869 \text{ Hz} \quad (115 - 4)$$

### RFEM 5 and RSTAB 8 Settings

- Modeled in RFEM 5.09.01 and RSTAB 8.09.01
- The element size is  $l_{FE} = 0.010 \text{ m}$
- The number of increments is 10
- Isotropic linear elastic model is used
- Root of the characteristic polynomial method is used

### Results

Structure Files	Program	Entity
0115.01	RFEM 5 – RF-DYNAM Pro	Member
0115.02	RSTAB 8 – DYNAM Pro	Member
0115.03	RFEM 5 – RF-DYNAM Pro	Plate
0115.04	RFEM 5 – RF-DYNAM Pro	Solid

Model	Analytical Solution	RFEM 5 / RSTAB 8	
	$f$ [Hz]	$f$ [Hz]	Ratio [-]
RFEM 5, Beam	4.869	4.868	1.000
RSTAB 8, Beam		4.868	1.000
RFEM 5, Plate		4.887	1.004
RFEM 5, Solid		4.889	1.004

### References

- [1] DLUBAL SOFTWARE GMBH, *Verification Example 0042 – Bending Cantilever with Axial Force*. 2015.
- [2] DLUBAL SOFTWARE GMBH, *Verification Example 0048 – Uniaxial Bending with Pressure*. 2016.