

**Program:** RFEM 5, RSTAB 8

**Category:** Large Deformation Analysis, Isotropic Linear Elasticity, Member

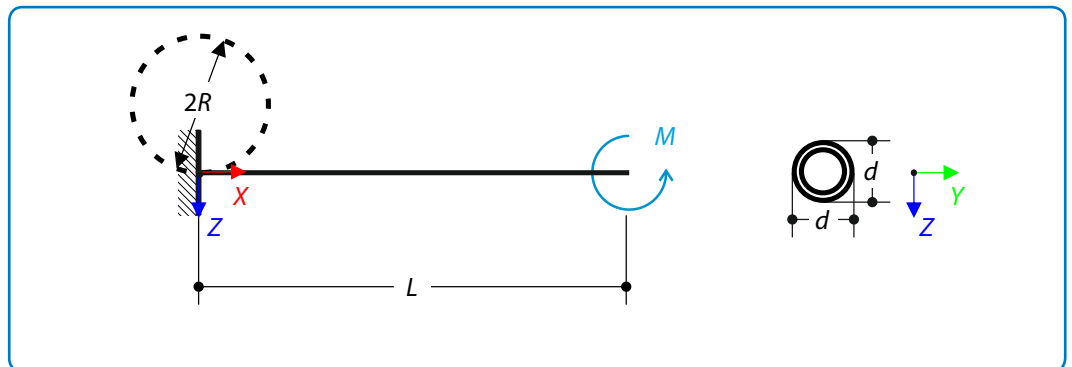
**Verification Example:** 0043 – Cantilever Bend to Form a Circle

## 0043 – Cantilever Bend to Form a Circle

### Description

Determine the bending moment  $M$ , which acts at the free end of the cantilever and which bends the member to a circular shape. Neglecting beam's self weight, assuming the large deformation theory and loading the cantilever with this particular  $M$ , check the maximum deflections  $u_{x,max}$  and  $u_{z,max}$ .

Material	Steel	Modulus of Elasticity	$E$	210.000	GPa
		Shear Modulus	$G$	81.000	GPa
Geometry	Beam	Length	$L$	4000.000	mm
		Diameter	$d$	42.400	mm
		Wall Thickness	$t$	4.000	mm



**Figure 1:** Problem sketch

### Analytical Solution

The second moment of inertia around y axis  $I_y$  equals to (see **Figure 1**):

$$I_y = \frac{\pi [d^4 - (d - 2t)^4]}{64} \approx 89908.5 \text{ mm}^4 \quad (43 - 1)$$

A beam in the large deformation analysis is described by the nonlinear differential equation

$$\kappa(x) = \frac{u_z''(x)}{[1 + (u_z'(x))^2]^{\frac{3}{2}}} = -\frac{M}{EI_y} \quad (43 - 2)$$

which is an equation difficult to solve in general. However, the term on the right-hand side is constant and consequently the left-hand side, which is nothing else then the beam curvature  $\kappa$ , is constant also. The only curve which has constant curvature is a circle, therefore, the solution to this problem is a circle arc of radius  $R$ . We get

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$$u_{x,\max} = R \sin \alpha - L \quad (43 - 3)$$

$$u_{z,\max} = R(1 - \cos \alpha) \quad (43 - 4)$$

where the radius of the circular arc equals to

$$R = \left| \frac{1}{\kappa(x)} \right| = \frac{EI_y}{M} \quad (43 - 5)$$

The angle of the circular arc equals to  $\alpha = \frac{L}{R}$ . In our case  $\alpha = 2\pi$ , which yields

$$R = \frac{L}{2\pi} \approx 636.620 \text{ mm} \quad (43 - 6)$$

The equations (43 – 5) and (43 – 6) yield the required loading moment

$$M = 2\pi \frac{EI_y}{L} \approx 29657.585 \text{ Nm} \quad (43 - 7)$$

Moreover, equations (43 – 3) and (43 – 4) yield the unknown maximum displacements

$$u_{x,\max} = -L = -4000.0 \text{ mm} \quad (43 - 8)$$

$$u_{z,\max} = -2R \approx -1273.2 \text{ mm} \quad (43 - 9)$$

### RFEM 5 and RSTAB 8 Settings

- Modeled in version RFEM 5.05.0030 and RSTAB 8.05.0030
- The element size is  $l_{FE} = 0.004 \text{ m}$
- The number of increments is 1
- Isotropic linear elastic material model is used
- Member division for large deformation or post-critical analysis is activated

### Results

Structure File	Program
0043.01	RFEM 5
0043.02	RSTAB 8

Good agreement of the numerical results with the analytical solution was achieved:

Displacement	Analytical Solution	RFEM 5		RSTAB 8	
	[mm]	[mm]	Ratio [-]	[mm]	Ratio [-]
$u_{x,\max}$	-4000.0	-3998.5	1.000	-4000.0	1.000
$u_{z,\max}$	-1273.2	-1273.1	1.000	-1273.2	1.000