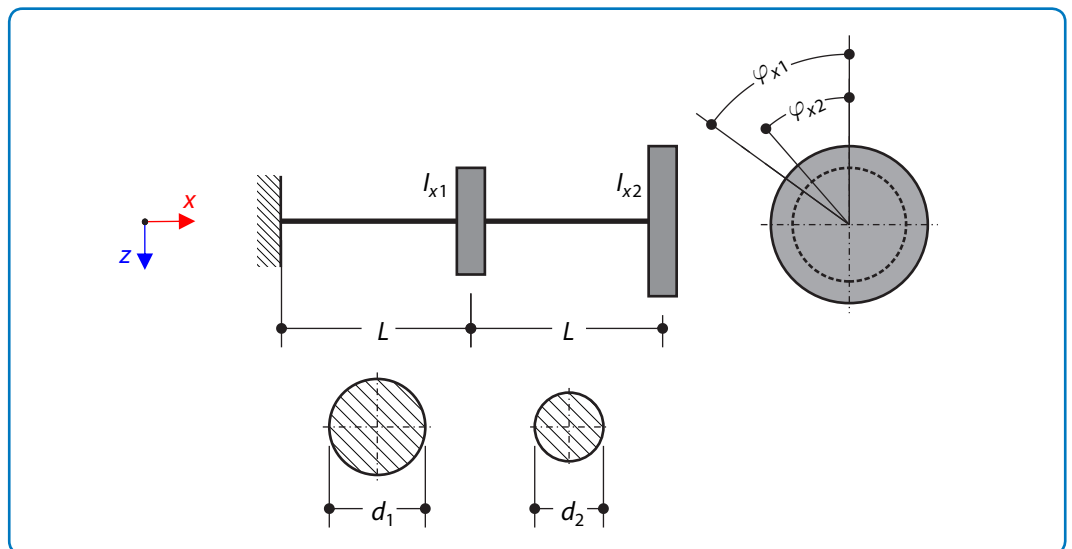


## 0111 – Torsional Vibrations

### Description

A double-mass system consists of two shafts of diameters  $d_1, d_2$  and two masses represented by the corresponding moments of inertia  $I_{x1}, I_{x2}$ , concentrated in given distance as nodal masses. The left shaft is fixed, and the right mass is free according to **Figure 1**. Neglecting the self-weight of the shafts, determine the torsional natural frequencies of the system (direction  $\varphi_x$ ). The problem is described by the following set of parameters.

Material	Steel	Modulus of Elasticity	$E$	210000.000	MPa
		Shear Modulus	$G$	81000.000	MPa
Geometry		Cross-section Diameter	$d_1$	20.000	mm
		Cross-section Diameter	$d_2$	40.000	mm
		Mass Moment of Inertia	$I_{x1}$	1.000	kgm <sup>2</sup>
		Mass Moment of Inertia	$I_{x2}$	0.700	kgm <sup>2</sup>
		Shaft Length	$L$	0.500	m



**Figure 1:** Problem Sketch

### Analytical Solution

The system of equations for the torsional motion, assuming that  $\varphi_{x1} > \varphi_{x2}$ , is described as

### Verification Example: 0111 – Torsional Vibrations

$$\mathbf{M}\ddot{\varphi}_x + \mathbf{K}\varphi_x = 0 \quad (111 - 1)$$

$$\begin{bmatrix} I_{x1} & 0 \\ 0 & I_{x2} \end{bmatrix} \begin{bmatrix} \ddot{\varphi}_{x1} \\ \ddot{\varphi}_{x2} \end{bmatrix} + \begin{bmatrix} k_1 & -k_1 \\ -k_1 & k_1 + k_2 \end{bmatrix} \begin{bmatrix} \varphi_{x1} \\ \varphi_{x2} \end{bmatrix} = 0 \quad (111 - 2)$$

where the torsional stiffnesses  $k_1, k_2$  are determined as

$$k_1 = \frac{\pi G d_1^4}{32L} \quad (111 - 3)$$

$$k_2 = \frac{\pi G d_2^4}{32L} \quad (111 - 4)$$

The eigenvalue problem

$$\det(\mathbf{K} - \Omega^2 \mathbf{M}) = 0 \quad (111 - 5)$$

yields the angular frequencies  $\Omega_1$  and  $\Omega_2$ , whence the natural frequencies  $f_1, f_2$  are calculated as follows

$$f_1 = \frac{\Omega_1}{2\pi} \approx 7.779 \text{ Hz} \quad (111 - 6)$$

$$f_2 = \frac{\Omega_2}{2\pi} \approx 39.615 \text{ Hz} \quad (111 - 7)$$

### RFEM 5 and RSTAB 8 Settings

- Modeled in RFEM 5.09.01 and RSTAB 8.09.01
- The element size is  $l_{FE} = 0.050 \text{ m}$
- The number of increments is 10
- Isotropic linear elastic model is used

### Results

Structure Files	Program	Member Type
0111.01	RFEM 5 - RF-DYNAM Pro	Beam
0111.02	RSTAB 8 - DYNAM Pro	Beam
0111.03	RFEM 5 - RF-DYNAM Pro	Definable Stiffness
0111.04	RSTAB 8 - DYNAM Pro	Definable Stiffness

**Verification Example: 0111 – Torsional Vibrations**

Model	Analytical Solution	RFEM 5 / RSTAB 8	
		$f_1$ [Hz]	Ratio [-]
RFEM 5, Beam	7.779	7.779	1.000
RFEM 8, Beam		7.779	1.000
RFEM 5, Definable Stiffness		7.779	1.000
RFEM 8, Definable Stiffness		7.779	1.000

Model	Analytical Solution	RFEM 5 / RSTAB 8	
		$f_2$ [Hz]	Ratio [-]
RFEM 5, Beam	39.615	39.615	1.000
RSTAB 8, Beam		39.615	1.000
RFEM 5, Definable Stiffness		39.615	1.000
RSTAB 8, Definable Stiffness		39.615	1.000